



Increase Factor of Safety of Go-Kart Chassis during Front Impact Analysis

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ABSTRACT

This paper aims to increase the factor of safety of the Go-Kart chassis which is designed keeping in mind the rules imposed Jaipur Street Karting Championship. Theoretical calculations are carried out which have been realized through several analyses. These result, coupled with appropriate research has been used to create a new chassis that possesses improved performance and safety. During front impact analysis, the chassis should meet the required factor of safety. In order to enhance factor of safety the computer aided design model was altered marginally such that it meets the safety requirements. An innovative method of design optimization has been discussed, without significant increase in the overall kerb weight of the chassis.

Key words: Chassis, Factor of Safety, Front Impact Analysis, Go-Kar

INTRODUCTION

Undercarriage or Chassis is a French expression and was at first used to mean the casing or principle structure of a vehicle [1]. Suspension is sub-isolated into the running rigging and the force plant. The running rigging incorporates the casing, directing framework, brakes, haggles tires. The power plant incorporates the motor gathering and force transmission get together [1] and henceforth body ought to have satisfactory quality to secure the driver in case of an effect. The suspension must have the ability to oppose all the sidelong and longitudinal powers applied on it. The structure and manufacture of the go - kart is cultivated remembering the prerequisites and rules forced by Go-Kart Configuration Challenge (GKDC) [2] which remember limitations for the vehicle's weight, shape, size and measurements. The skeleton must be displayed considering the dynamic occasions in GKDC which incorporate increasing speed test, brake test, autocross, slide cushion and perseverance test which would test a definitive execution of the suspension on the track. Keeping these occasions and the obliges of the standard book at the top of the priority list, five frame were displayed utilizing CATIA V5 R20 and basic investigation was performed on these undercarriage utilizing ANSYS 14. The case that created a good outcome was picked for structure streamlining so as to expand the factor of wellbeing.

The vulnerabilities related with the hardware concerning pressure and quality incorporates:

- Creation of material and the impact of minor departure from properties.
- Varieties in properties here and there inside a bar of stock.
- Impact of close by gatherings, for example, weld ments and therapist fits on pressure conditions

We should oblige vulnerability [3]. There are scientific strategies to address vulnerability, the essential method being the deterministic strategy. The deterministic technique set up is the plan factor dependent on the supreme vulnerabilities of a misfortune of capacity boundary and a most extreme suitable boundary can be load, stress, redirection, and so forth [3]. In this manner, factor of security can be characterized as the proportion of the yield worry to the working worry for a flexible material.

MATERIAL

The Material picked for displaying the suspension is AISI 1018. AISI 1018 is a gentle/low carbon steel offering a decent parity of durability, quality and pliability [4]. Alongside the previously mentioned properties, the material additionally has great weld ability view points. In this way, AISI 1018 was picked for examination and manufacture.

Round about consistent cylinders having an external distance across (OD) of 0.02921m (1.15 inches), with a divider thickness of 0.002 m (0.078 inches) was chosen. The mechanical properties of the material are appeared in table 1.

Table –1 Mechanical Properties of AISI 1018

Property	Estimation
Thickness	7.87 g/cc
Rigidity (Yield)	370 MPa
Rigidity (Extreme)	440 MPa
Modulus of Versatility	205 GPa
Mass Modulus	40 GPa
Poisson's Proportion	0.29
Shear Modulus	80 GPa

Substance structures of the material are appeared in table 2.

Table –2 Substance Structure of AISI 1018

Structure Rate (%)	Structure Rate (%)
Carbon (C)	0.14 – 0.20
Iron (Fe)	98.81 – 99.26
Manganese (Mn)	0.60 – 0.90
Phosphorus (P)	<= 0.04
Sulphur (S)	<= 0.05

CHASSIS DESIGN

Our essential centre was to build load-bearing capacities, deteriorate the expense, to diminish weight of body and strengthen driver's wellbeing. The body which created results meeting the wellbeing and burden bearing necessities was picked for additional investigation, which is appeared in Fig 1a.

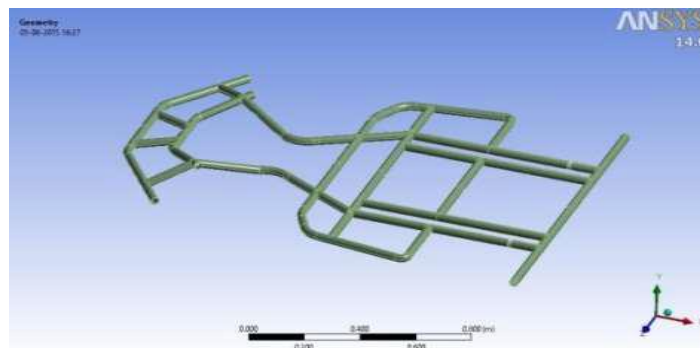


Fig. 1a Displayed body that is utilized for additional investigation.

The demonstrated Undercarriage weighs 16.64 kg.

Whole frame was coincided as customary tetrahedron cells with sides of length 0.005 m. Hypothetical determined burdens were applied at various parts of the body. Fig 1b shows the cross section of body.

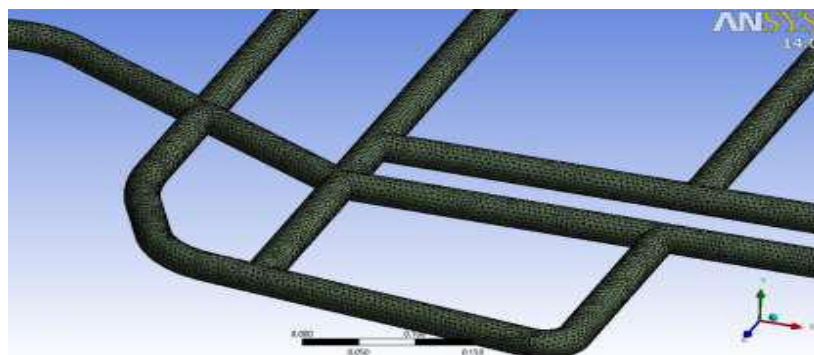


Fig. 1b Normal tetrahedron fit suspension.

FRONT EFFECT EXAMINATION

So as to perform Front Effect Investigation, the back guard of the Suspension was obliged. The power is applied at the front guard which is determined utilizing Newton's mass second condition,

$$F = m \times \delta v / t. \tag{1}$$

Where,

m- mass of the vehicle with the driver.

δv - change in speed.

t- time taken to decelerate.

The complete mass of the vehicle is 165 kg (incorporates weight of the motor, drive framework, brake unit, body packs and weight of the driver) and all out time taken for the Go-Kart to decelerate from 60-0km/hr and stop is 0.85seconds.

Subsequently, the power was determined utilizing (1)

$$F = 165 \times 16.67 / 0.85$$

$$F = 3235.94 \text{ N}$$

RESULTS

Table 3 delineates the Pressure, Strain and All out Disfigurement.

Table -3 Examination Results

Type	Maximum	Minimum
Stress	1.309e8 Pa	3.725e-10 Pa
Strain	0.000696	3.055e-15
Total Deformation	0.015714 m	0 m

The factor of wellbeing (FOS) acquired during front effect investigation is 1.99. Mechanical properties accessible for us are expected to uniaxial testing, the frame could bomb because of other sidelong loads following up on it during its exhibition on the track and consequently FOS for the suspension ought to be more prominent than 2 for the plan to be sheltered. It is obvious from Fig 2a that the skeleton is exposed to disappointment at the parallel twists, since suspension is exceptionally focused.

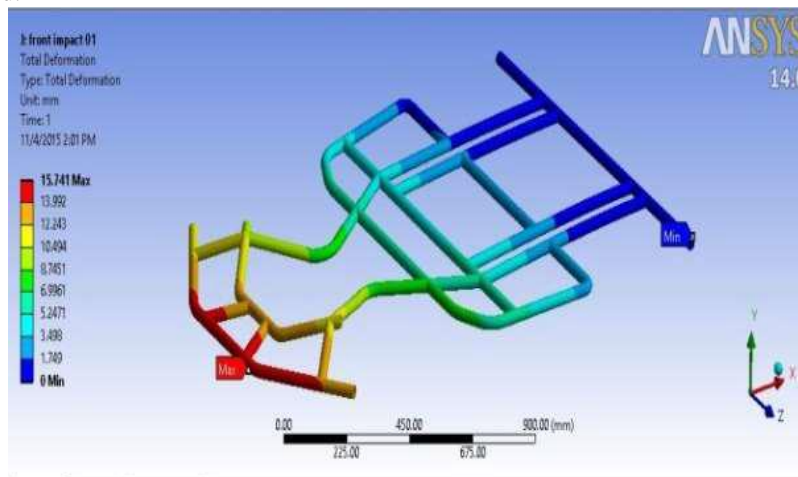


Fig. 2a

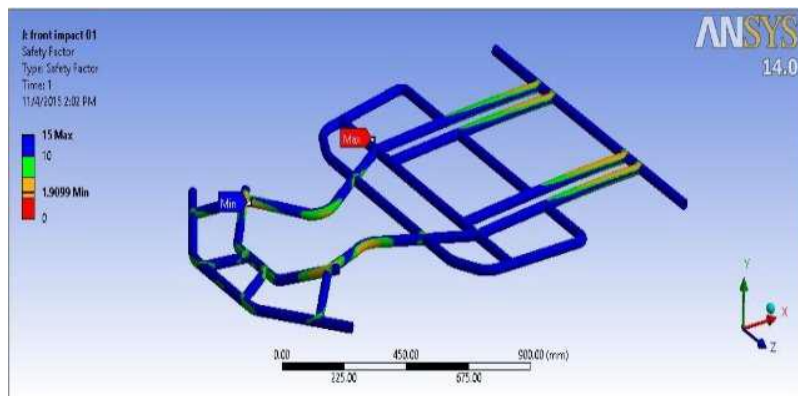


Fig. 2b

Fig. 2 Results acquired after investigation (a) Complete miss happening (b) Factor of Wellbeing.

STRUCTURE OPTIMIZATION

So as to build the FOS of the suspension and produce a steady structure during front effect, an AISI 1018 shrub having the following measurements was utilized:

- External distance across of 0.0385m,
- Internal width of 0.0305m, and
- 0.050m long.

The demonstrated mellow steel brambles appeared in Fig 3a are so structured to such an extent that they can be embedded into the front guard of the suspension and in this way having a freedom fit. The other supporting structures of the guard and the skeleton are welded on the shrub which is appeared in Fig 3b.

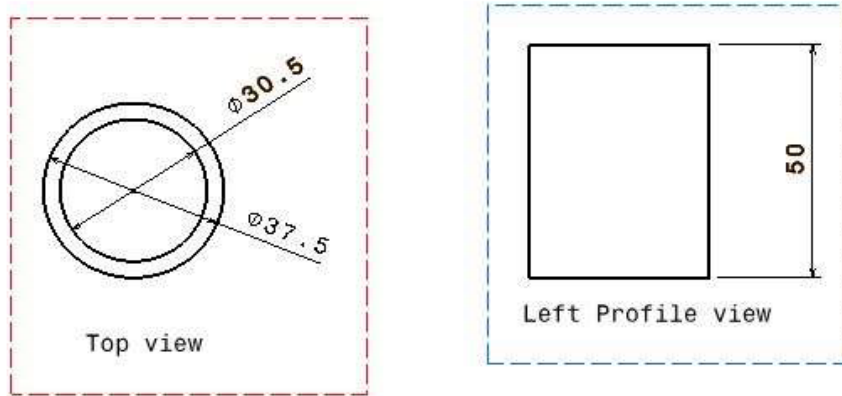


Fig. 3a Structured shrubbery measurements in mm.

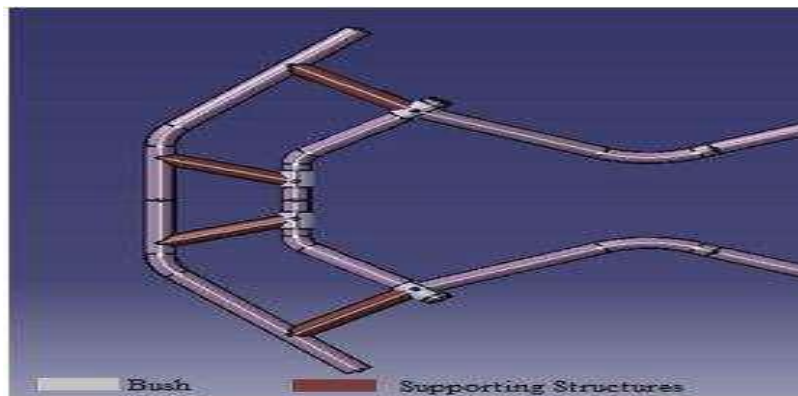


Fig. 3b Utilization of shrubbery on the body

During Effect

During time of impact the power is applied on to the front guard where vitality is moved from the guard to the body through the supporting structure and the shrubbery. At the hour of crash, the guard assimilates 20% of the all out effect vitality and the staying 80% of the vitality is moved onto the shrubbery. As the shrubbery have a leeway fit with the front guard, the sway power is changed over into rotational power about the pivot of the shrubbery, and consequently diminishing the heap move on the body. Examination was performed on the shrub were a power of 2588.752 N was applied.

Results

Table 4 portrays the Pressure, Strain and Complete Distortion on the investigation performed on the hedge.

Table –4 Investigation After effects of the bush

Type	Maximum	Minimum
Stress	1.69e7 Pa	0.00089 e6Pa
Strain	9.2196e-5	5.0134e-9
Total Distortion	0.00153mm	0.mm

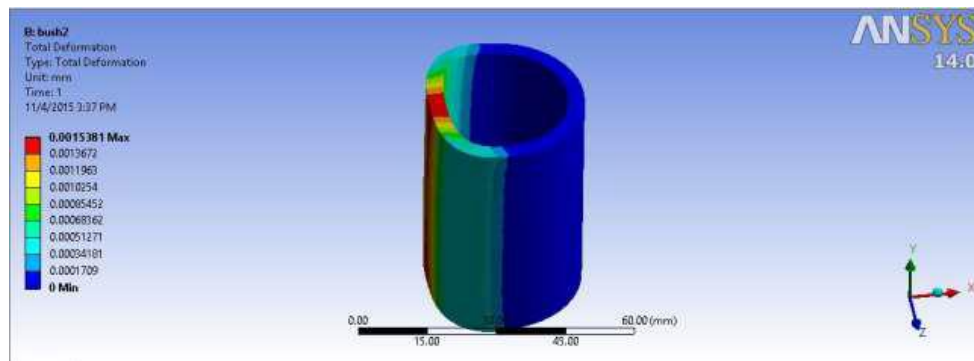


Fig. 4a All out Distortion of the hedge

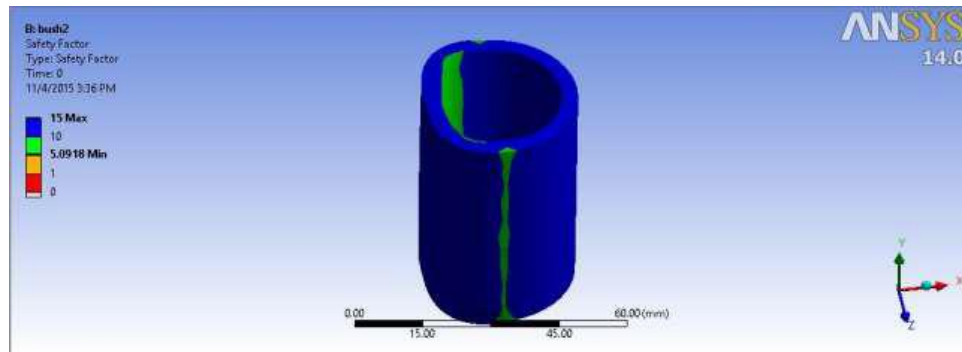


Fig. 4b FOS got after examination on the bramble Factor of wellbeing for the shrubbery after investigation was 5.0918

CONCLUSION

Go-Karts similarity has been examined in numerous examinations utilizing various methodologies, for example, certifiable accident insights and crash testing, however because of lacking assets, the previously mentioned methods can't be adjusted. With the hypothetical methodology of investigations subsequent to embeddings the shrub it is seen that the FOS of the shrubbery alone is 5.0918 which is appeared in Fig 4b.

Coupling both the FOS of suspension just as the bramble the general FOS builds in excess of 2 in this manner demonstrating that the body is safe during front effect and the vulnerabilities during the structure stage are taken consideration. During the effect it is additionally observed that the geometry of the body stays unaltered without influencing the Karts execution on the track.

Hypothetical outcomes got can change as per genuine situation.

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